



Nom :

Prénom :

Signature du candidat	Numéro d'inscription	Matricule
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N° Anonymat

N° Anonymat

CONCOURS NATIONAL D'INGÉNIEURS DE MAURITANIE (CNIM)



CONCOURS 2017

ÉPREUVE DES SCIENCES DE L'INGENIEUR

Durée de l'épreuve : 3 heures

**L'usage de l'ordinateur, de la calculatrice ou
tout autre objet connecté est interdit.**

Le sujet comporte:

- 11 pages d'énoncé
- 12 pages de document réponse

Si au cours de l'épreuve, le candidat repère ce qui lui semble être une erreur d'énoncé, il le signale sur sa copie et poursuit sa composition en expliquant les raisons des initiatives qu'il est amené à prendre.

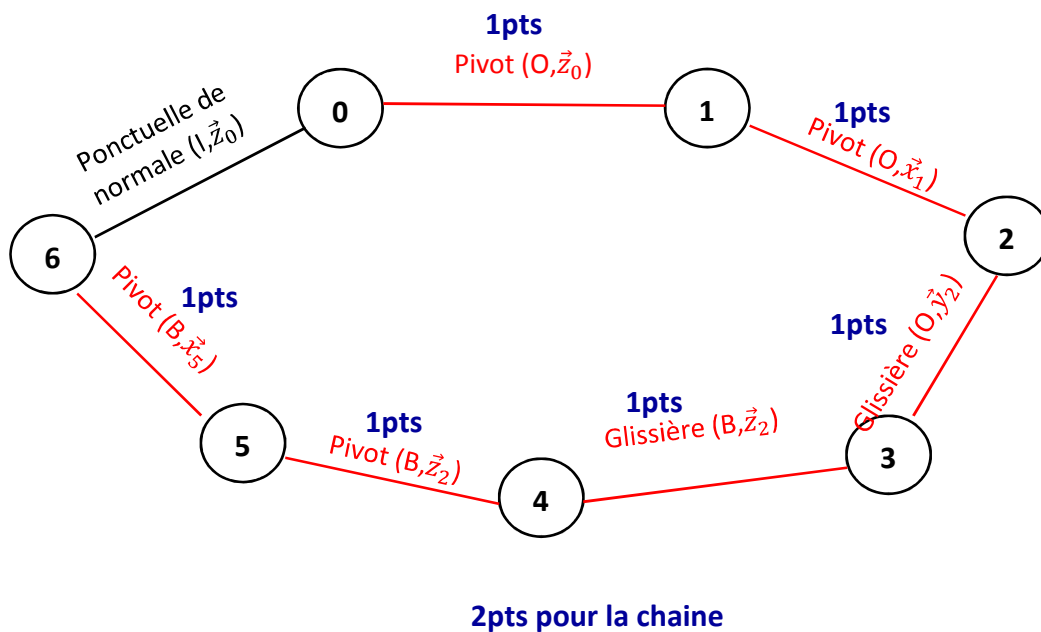
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NE RIEN ECRIRE ICI

Partie 1. Validation des capacités d'adaptation de la passerelle à son environnement

Question 1.

8pts



Question 2.

8pts

$$\vec{y}_2 = \cos \beta \vec{y}_1 + \sin \beta \vec{z}_0 = \cos \beta (-\sin \alpha \vec{x}_0 + \cos \alpha \vec{y}_0) + \sin \beta \vec{z}_0$$

$$\vec{y}_2 = -\sin \alpha \cos \beta \vec{x}_0 + \cos \alpha \cos \beta \vec{y}_0 + \sin \beta \vec{z}_0$$

$$\vec{z}_2 = -\sin \beta \vec{y}_1 + \cos \beta \vec{z}_0 = -\sin \beta (-\sin \alpha \vec{x}_0 + \cos \alpha \vec{y}_0) + \cos \beta \vec{z}_0$$

$$\vec{z}_2 = \sin \alpha \sin \beta \vec{x}_0 - \cos \alpha \sin \beta \vec{y}_0 + \cos \beta \vec{z}_0$$

4pts

4pts

$$\vec{y}_2 = \begin{bmatrix} \dots - \sin \alpha \cos \beta \dots \\ \dots \cos \beta \cos \alpha \dots \\ \dots \sin \beta \dots \end{bmatrix}_{R_0}$$

$$\vec{z}_2 = \begin{bmatrix} \dots \sin \beta \sin \alpha \dots \\ \dots -\sin \beta \cos \alpha \dots \\ \dots \cos \beta \dots \end{bmatrix}_{R_0}$$

Question 3.

8pts

$$\vec{OI} = \vec{OA} + \vec{AB} + \vec{BI}$$

$$x_I \vec{x}_0 + y_I \vec{y}_0 + z_I \vec{z}_0 = \lambda(t) \vec{y}_2 - \mu(t) \vec{z}_2 - \frac{D}{2} \vec{z}_0 \quad \text{2pts}$$

$$\begin{pmatrix} x_I \\ y_I \\ z_I \end{pmatrix}_{R_0} = \lambda(t) \begin{pmatrix} -\sin \alpha \cos \beta \\ \cos \beta \cos \alpha \\ \sin \beta \end{pmatrix}_{R_0} - \mu(t) \begin{pmatrix} \sin \beta \sin \alpha \\ -\sin \beta \cos \alpha \\ \cos \beta \end{pmatrix}_{R_0} - \frac{D}{2} \begin{pmatrix} 0 \\ 0 \\ 1 \end{pmatrix}_{R_0}$$

$$\begin{cases} x_I = -\lambda(t) \sin \alpha \cos \beta - \mu(t) \sin \beta \sin \alpha \\ y_I = \lambda(t) \cos \beta \cos \alpha + \mu(t) \sin \beta \cos \alpha \\ z_I = \lambda(t) \sin \beta - \mu(t) \cos \beta - \frac{D}{2} \end{cases}$$

$$\begin{cases} x_I = \dots - \lambda(t) \sin \alpha \cos \beta - \mu(t) \sin \beta \sin \alpha \dots \dots \dots (1) & \text{2pts} \\ y_I = \dots \dots \dots \lambda(t) \cos \beta \cos \alpha + \mu(t) \sin \beta \cos \alpha \dots \dots \dots (2) & \text{2pts} \\ z_I = \dots \dots \dots \lambda(t) \sin \beta - \mu(t) \cos \beta - \frac{D}{2} \dots \dots \dots (3) & \text{2pts} \end{cases}$$

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Question 4.

8pts

$$\vec{ID} = \vec{IB} + \vec{BA} + \vec{AD}$$

2pts

$$x_D \vec{x}_0 + y_D \vec{y}_0 + z_D \vec{z}_0 = \frac{D}{2} \vec{z}_0 + \mu(t) \vec{z}_2 + l \vec{y}_2$$

$$\begin{pmatrix} x_D \\ y_D \\ z_D \end{pmatrix}_{R_0} = \frac{D}{2} \begin{pmatrix} 0 \\ 0 \\ 1 \end{pmatrix}_{R_0} + \mu(t) \begin{pmatrix} \sin \beta \sin \alpha \\ -\sin \beta \cos \alpha \\ \cos \beta \end{pmatrix}_{R_0} + l \begin{pmatrix} -\sin \alpha \cos \beta \\ \cos \beta \cos \alpha \\ \sin \beta \end{pmatrix}_{R_0}$$

$$\begin{cases} x_D = \mu(t) \sin \beta \sin \alpha - l \sin \alpha \cos \beta \\ y_D = -\mu(t) \sin \beta \cos \alpha + l \cos \beta \cos \alpha \\ z_D = \frac{D}{2} + \mu(t) \cos \beta + l \sin \beta \end{cases}$$

$$x_D = \dots \mu(t) \sin \beta \sin \alpha - l \sin \alpha \cos \beta \dots (4)$$

2pts

$$y_D = \dots -\mu(t) \sin \beta \cos \alpha + l \cos \beta \cos \alpha \dots (5)$$

2pts

$$z_D = \dots \frac{D}{2} + \mu(t) \cos \beta + l \sin \beta \dots (6)$$

2pts

Question 5.

8pts

$$\alpha = 0 \rightarrow x_D = 0 \text{ et } x_I = 0 \quad \mathbf{1pts}$$

$$y_I = \lambda(t) \cos \beta + \mu(t) \sin \beta = y_F \quad \mathbf{1pts}$$

$$z_I = \lambda(t) \sin \beta - \mu(t) \cos \beta - \frac{D}{2} = -h \quad \mathbf{1pts}$$

$$y_D = -\mu(t) \sin \beta + l \cos \beta = y_a \quad \mathbf{1pts}$$

$$z_D = \frac{D}{2} + \mu(t) \cos \beta + l \sin \beta = z_a \quad \mathbf{1pts}$$

A partir de ce système, on a :

$$z_a - h = (l + \lambda(t)) \sin \beta \quad \mathbf{1pts}$$

$$y_a + y_F = (l + \lambda(t)) \cos \beta \quad \mathbf{1pts}$$

D'où :

$$\tan \beta = \frac{z_a - h}{y_a + y_F} \quad \mathbf{1pts}$$

Question 6.**8pts**

$$\tan \beta = \frac{z_a - h}{y_a + y_F} = \frac{2 - 4}{38 + 2} = -5\% \quad \text{4pts}$$

Le cahier des charges est respecté **4pts**

Partie 2. Validation de la charge maximale admissible par l'essieu de la passerelle

Question 7.**8pts**

$$\vec{OG} = \frac{1}{\sum_i m_i} \sum_i m_i \vec{OG}_i \quad \text{4pts}$$

$$y_G = \frac{m_2 y_2 + m_3 y_3}{m_2 + m_3} \quad \text{4pts}$$

Question 8.**8pts**

$$\vec{S} = \{0, 4, g\}$$

$$\{F(0 \rightarrow S)\} = \begin{array}{c} \boxed{X} \\ \left. \begin{array}{l} 0 \\ Y \cdot M \\ Z \cdot N \end{array} \right\} \end{array} \quad \text{2pts} \quad R_0$$

$$\vec{M}_O(4 \rightarrow S) = \vec{M}_B(4 \rightarrow S) + \vec{R}(4 \rightarrow S) \wedge \vec{BO} = F \vec{z}_2 \wedge (a \vec{z}_2 - b \vec{y}_2) = bF \vec{x}_0$$

$$\{F(4 \rightarrow S)\} = \begin{array}{c} \boxed{0} \\ \left. \begin{array}{l} F \sin \beta \\ F \cos \beta \end{array} \right\} \end{array} \begin{array}{l} bF \\ 0 \\ 0 \end{array} \quad \text{2pts} \quad R_0$$

$$\vec{M}_O(g \rightarrow S) = \vec{OG} \wedge M \vec{g} = y_G \vec{y}_2 \wedge (-Mg \vec{z}_0) = -Mg y_G \cos \beta \vec{x}_0$$

$$\{F(4 \rightarrow S)\} = \begin{array}{c} \boxed{0} \\ \left. \begin{array}{l} 0 \\ -Mg \end{array} \right\} \end{array} \begin{array}{l} -Mg y_G \cos \beta \\ 0 \\ 0 \end{array} \quad \text{2pts} \quad R_0$$

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$$\{F(\bar{S} \rightarrow S)\} = \left\{ \begin{array}{l} \dots X \dots \dots \\ \dots Y + F \sin \beta \dots \dots \\ \dots Z + F \cos \beta - Mg \dots \dots \end{array} \right\}_{R_0} \left\{ \begin{array}{l} \dots bF - Mgy_G \cos \beta \dots \\ \dots M \dots \\ \dots N \dots \dots \end{array} \right\} \quad \mathbf{2pts}$$

Question 9.

8pts

$$\vec{x}_0 \cdot \vec{\mathcal{M}}_0(\bar{S} \rightarrow S) = 0 \quad \mathbf{4pts}$$

$$bF - Mgy_G \cos \beta$$

$$F = \frac{Mgy_G \cos \beta}{b} \quad \mathbf{4pts}$$

Question 10.

8pts

$$F = \frac{Mgy_G \cos \beta}{b} = \frac{60 \cdot 10^3 \times 10 \times 10}{15} = 4 \cdot 10^5 N = 40 \text{ tonnes} \quad \mathbf{4pts}$$

$$F = 40 \text{ tonnes} < 70 \text{ tonnes} \quad \mathbf{2pts}$$

Le cahier des charges est respecté $\mathbf{2pts}$

Partie 3. Validation de l'aptitude de la passerelle au déplacement horizontal

Question 11.

8pts

$$\vec{V}(I \in 6/sol) = \vec{V}(B \in 6/sol) + \vec{\Omega}(6/sol) \wedge \vec{BI} = \dot{y}(t) \vec{y}_0 + \left(\frac{-\omega_m(t)}{K_r} \vec{x}_0 \right) \wedge \left(\frac{-D}{2} \vec{z}_\Omega \right)$$

$$\vec{V}(I \in 6/sol) = \left(\dot{y}(t) - \frac{D\omega_m(t)}{2K_r} \right) \vec{y}_0 \quad \mathbf{2pts}$$

Condition de roulement sans glissement : $\vec{V}(I \in 6/sol) = \vec{0}$

$$\dot{y}(t) = \frac{D\omega_m(t)}{2K_r} \quad \mathbf{2pts}$$

$$\omega_{max} = \frac{2K_r}{D} \dot{y}_{max} \quad \mathbf{2pts}$$

$$AN : \omega_{max} = \frac{2 \times 2000}{1} \times 1,2 = 4800 \text{ rad/s} = 12,73 \text{ tr/min} \quad \mathbf{2pts}$$

Question 12.**8pts**

$$\Sigma = \{3,6\}$$

$$E_c(3/0) = \frac{1}{2} m_3 \dot{y}^2 = \frac{1}{2} m_3 \frac{D^2}{4K_r^2} \omega_m^2(t) \quad \mathbf{2pts}$$

$$E_c(6/0) = \frac{1}{2} m_6 \dot{y}^2 + \frac{1}{2} A_6 \dot{\theta}^2 = \frac{1}{2} m_6 \frac{D^2}{4K_r^2} \omega_m^2(t) + \frac{1}{2} A_6 \frac{\omega_m^2(t)}{K_r^2} = \frac{1}{2} \frac{\omega_m^2(t)}{K_r^2} \left(m_6 \frac{D^2}{4} + A_6 \right) \quad \mathbf{2pts}$$

$$E_c(\Sigma/0) = \frac{1}{2} \frac{\omega_m^2(t)}{K_r^2} \left((m_3 + m_6) \frac{D^2}{4} + A_6 \right) = \frac{1}{2} J_{eq} \omega_m^2(t) \text{ avec } J_{eq} = \frac{1}{K_r^2} \left((m_3 + m_6) \frac{D^2}{4} + A_6 \right)$$

$$E_c(\Sigma/R_0) = \frac{1}{2} J_{eq} \omega_m^2(t) \quad \mathbf{2pts}$$

$$J_{eq}(\Sigma) = \frac{1}{K_r^2} \left((m_3 + m_6) \frac{D^2}{4} + A_6 \right) \quad \mathbf{2pts}$$

Question 13.**8pts**

$P(\text{int à } \Sigma) = P(6 \leftrightarrow 3) = 0$ car liaison parfaite.

8pts

$$P(\text{int à } \Sigma) = 0$$

Question 14.**8pts**

$$\Sigma = \{3,6\} \rightarrow \bar{\Sigma} = \{0, \text{sol}, g, MR_1\} \quad \mathbf{1pts}$$

$$P(\bar{\Sigma} \rightarrow \Sigma/R_0) = P(0 \rightarrow 3/R_0) + P(\text{sol} \rightarrow 6/R_0) + P(g \rightarrow 3/R_0) + P(g \rightarrow 6/R_0) + P(MR_1 \rightarrow 6/R_0) \quad \mathbf{1pts}$$

$$P(\text{sol} \rightarrow 6/R_0) = 0 \text{ car condition de roulement sans glissement} \quad \mathbf{1pts}$$

$$P(g \rightarrow 3/R_0) = P(g \rightarrow 6/R_0) = 0 \quad \mathbf{2pts}$$

$$P(MR_1 \rightarrow 6/R_0) = \eta c_m(t) \omega_m(t)$$

$$P(0 \rightarrow 3/R_0) = -f_R(t) \dot{y}(t) = -\frac{D}{2K_r} f_R(t) \omega_m(t) \quad \mathbf{1pts}$$

$$P(\bar{\Sigma} \rightarrow \Sigma/R_0) = \left(\eta c_m(t) - \frac{D}{2K_r} f_R(t) \right) \omega_m(t) \quad \mathbf{1pts}$$

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$$P(\bar{\Sigma} \rightarrow \Sigma/R_0) = \left(\eta c_m(t) - \frac{D}{2K_r} f_R(t) \right) \omega_m(t) \quad \mathbf{1pts}$$

Question 15.

8pts

$$\frac{d\dot{E}_c(\bar{\Sigma}/R_0)}{dt} = P(\bar{\Sigma} \rightarrow \Sigma/R_0) + P(\text{int à } \Sigma) \quad \mathbf{4pts}$$

$$J_{eq} \omega_m(t) \frac{d\omega_m(t)}{dt} = \omega_m(t) \left(\eta c_m(t) - \frac{D}{2K_r} f_R(t) \right)$$

$$c_m(t) = \frac{1}{\eta} \left(J_{eq} \frac{d\omega_m(t)}{dt} + \frac{D}{2K_r} f_R(t) \right)$$

$$c_m(t) = \frac{1}{\eta} \left(J_{eq} \frac{d\omega_m(t)}{dt} + \frac{D}{2K_r} f_R(t) \right) \quad \mathbf{4pts}$$

Question 16.

8pts

Période	$[0, T_a]$	$]T_a, T - T_a[$	$[T - T_a, T]$
$\frac{d\omega_m(t)}{dt}$ en $(rads^{-2})$	$\frac{\omega_{max}}{T_a}$ 1pts	1pts 0	1pts $-\frac{\omega_{max}}{T_a}$
Couple moteur $c_m(Nm)$ Expression analytique	$\frac{1}{\eta} \left(J_{eq} \frac{\omega_{max}}{T_a} + \frac{D}{2K_r} f_R(t) \right)$ 1pts	$\frac{D}{2\eta K_r} f_R(t)$ 1pts	$\frac{1}{\eta} \left(-J_{eq} \frac{\omega_{max}}{T_a} + \frac{D}{2K_r} f_R(t) \right)$ 1pts

Le moteur fournit un couple maximal dans la période $[0, T_a]$ **2pts**

Partie 4. Validation des performances de déplacement horizontal de la passerelle

Question 17.

8pts

$U(p) = RI(p) + E(p)$ **2pts**

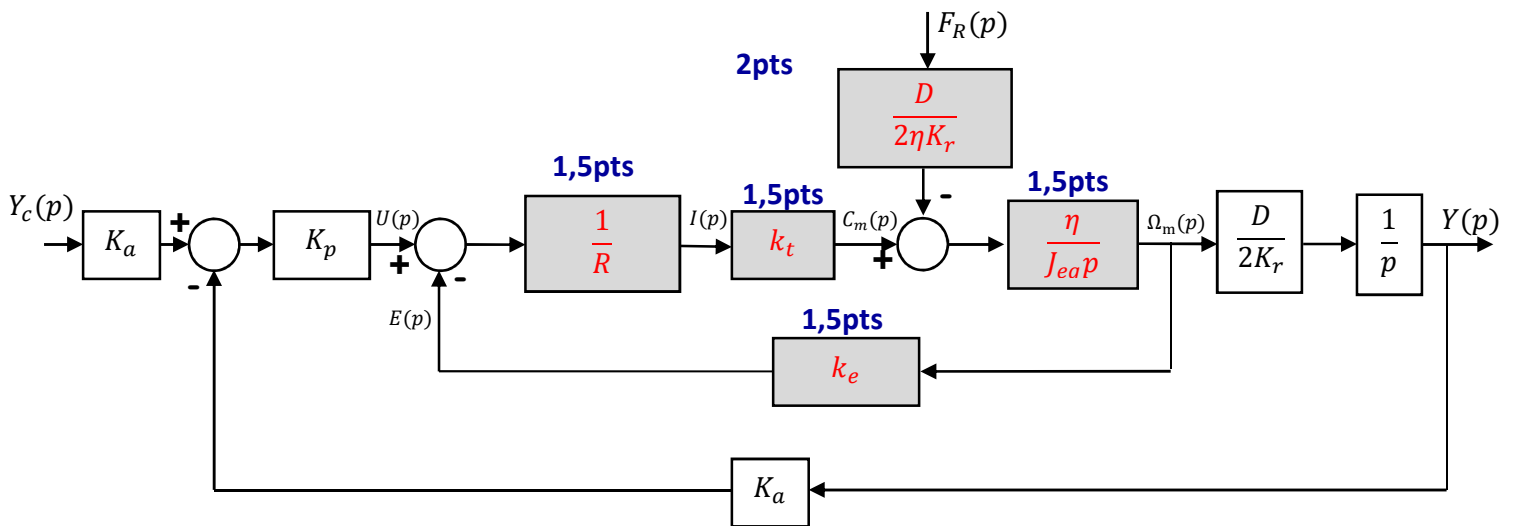
$E(p) = k_e \Omega_m(p)$ **2pts**

$C_m(p) = k_t I(p)$ **2pts**

$C_m(p) = \frac{D}{2\eta K_r} F_R(p) = \frac{J_{eq}}{\eta} p \Omega_m(p)$ **2pts**

Question 18.

8pts



Question 19.

8pts

$H_1(p) = \frac{Y(p)}{Y_c(p)} = \frac{1}{1 + \frac{2k_r k_e}{k_p k_a D} p + \frac{2k_r R J_{eq}}{\eta k_p k_a k_t D} p^2}$ **8pts**

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$$H_1(p) = \frac{Y(p)}{Y_c(p)} = \frac{1}{1 + \frac{2K_r k_e}{k_p k_a D} p + \frac{2K_r R J_{eq}}{\eta k_p k_a k_t D} p^2}$$

Question 20.

8pts

$$H_1(p) = \frac{1}{1 + \frac{2K_r k_e}{k_p k_a D} p + \frac{2K_r R J_{eq}}{\eta k_p k_a k_t D} p^2} = \frac{k_s}{1 + \frac{2m}{\omega_0} p + \frac{p^2}{\omega_0^2}}$$

$$k_s = 1$$

$$\frac{1}{\omega_0^2} = \frac{2K_r R J_{eq}}{\eta k_p k_a k_t D} \rightarrow \omega_0 = \sqrt{\frac{\eta k_p k_a k_t D}{2K_r R J_{eq}}}$$

$$\frac{2m}{\omega_0} = \frac{2K_r k_e}{k_p k_a D} \rightarrow m = k_e \sqrt{\frac{\eta k_r k_t}{2k_p k_a D R J_{eq}}}$$

2pts

3pts

3pts

$$k_s = 1$$

$$\omega_0 = \sqrt{\frac{\eta k_p k_a k_t D}{2K_r R J_{eq}}}$$

$$m = k_e \sqrt{\frac{\eta k_r k_t}{2k_p k_a D R J_{eq}}}$$

Question 21.

8pts

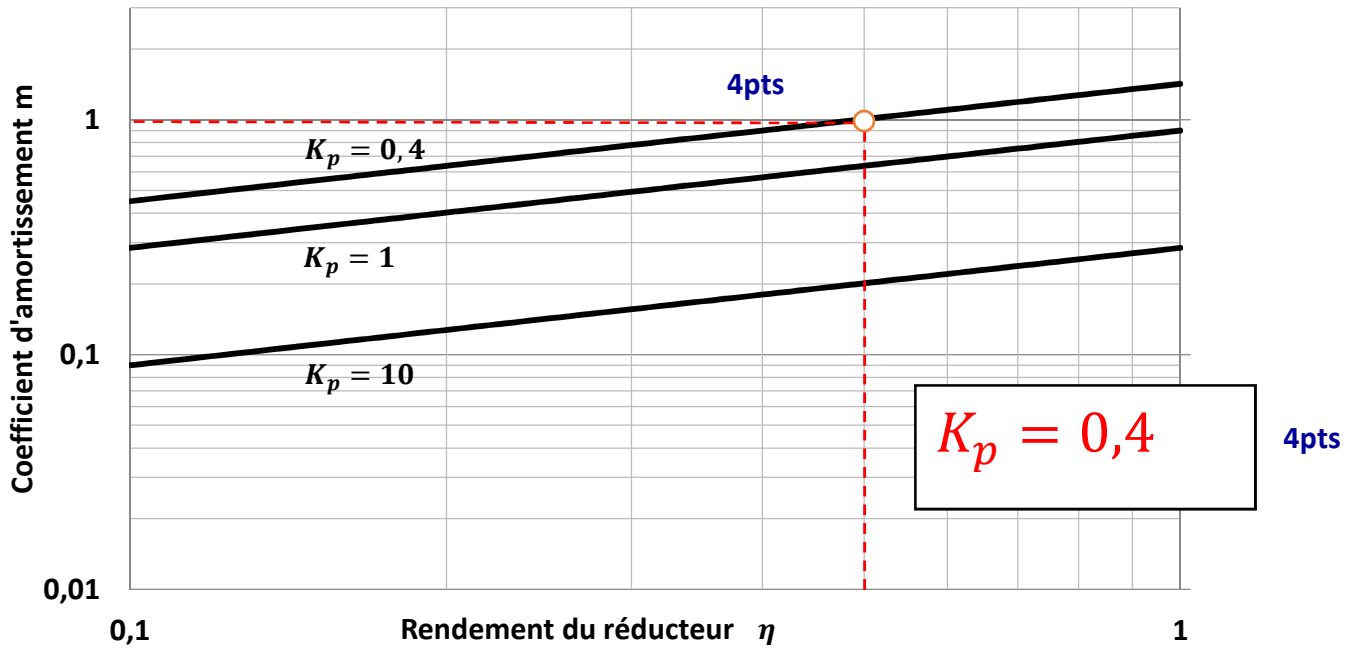
Réponse rapide sans dépassement $\rightarrow m = 1$ 4pts

$$m = k_e \sqrt{\frac{\eta k_r k_t}{2k_p k_a D R J_{eq}}} = 1 \rightarrow k_p = \frac{\eta k_r k_t k_e^2}{2k_a D R J_{eq}}$$
 4pts

$$k_p = \frac{\eta k_r k_t k_e^2}{2k_a D R J_{eq}}$$

Question 22.

8pts



Question 23.

8pts

$$FTBF(p) = \frac{0,0625k_p}{0,0625k_p + p + 10p^2} \quad 4pts$$

$$FTBF(p) = \frac{0,0625k_p}{0,0625k_p + p + 10p^2}$$

$$D(p) = 0,0625k_p + p + 10p^2$$

Le système est stable ssi $k_p > 0$ 4pts

Question 24.

8pts

$$t_{r5\%} = 94s \quad 1pts$$

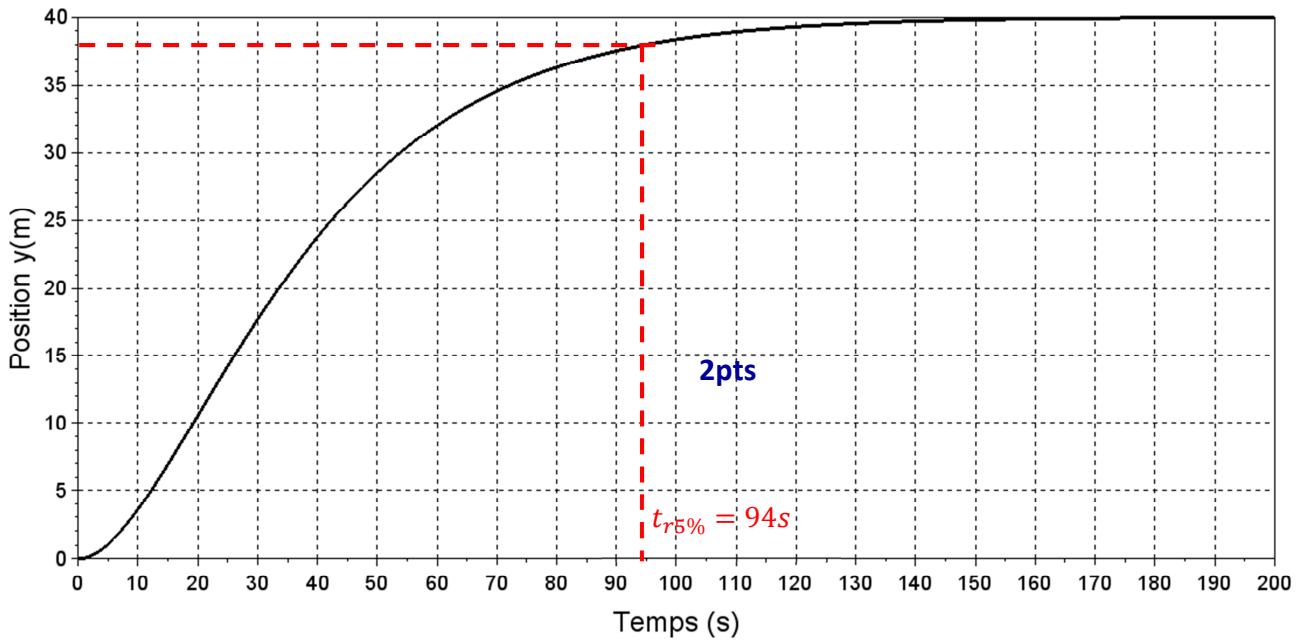
$$\varepsilon_p = 0 \quad 1pts$$

Conclusion :

Critère de précision respecté 2pts

Critère de rapidité non respecté 2pts

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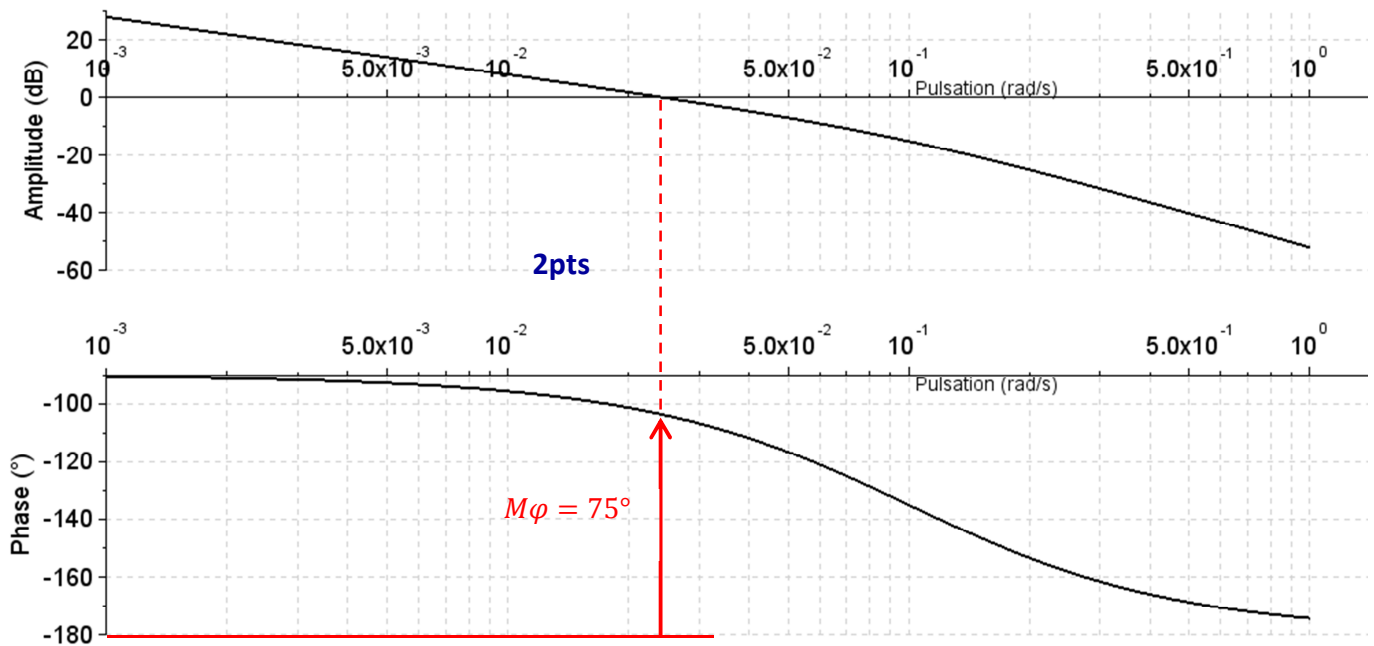
Question 25.

8pts

Marge de phase = $75^\circ > 45^\circ$ 2pts

Marge de gain. = +infini. 2pts

Critère de stabilité vérifié 2pts



*** FIN ***